SPACE SHUTTLE ORBITER THRUST STRUCTURE

DESIGN PROJECT 1 Final Report AEE 471 | Davidson Assigned:August 31st, 2018 Due Date: September 28th, 2018

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PROVIDED INFORMATION

This project encompasses the design of a space shuttle orbiter upper thrust structure. This structure is simplified and shown below in *Figure 1*. The end goal of this project is to appropriately design the idealized upper thrust structure as seen in *Figure 1* featuring minimum weights.



Each engine (with 3 in total) generates 375,000 lbf of thrust and can gimbal up to 8° . The forces *P1*, *P2*, and *P3* are shown below for each case, where cases 2 and 3 account for anticipated gimbal.

	CASE 1	CASE 2	CASE 3
P1	101,196 lbf	120,000 lbf	88,391 lbf
Р2	227,655 lbf	227,655 lbf	227,655 lbf
Р3	104,196 lbf	88,391 lbf	120,000 lbf

Table 1 (Davidson, AEE 471 Handout, 8/31/18)

Each beam will be designed from Ti-5AL-2.5Sn Titanium and B4/N5505 Boron/Epoxy, which feature the following properties:

	Titanium	Boron
Modulus, E (msi)	15.500	29.600
Poisson's ratio v	0.330	0.210
Yield stress, σ_{ys} (ksi)	110.000	n/a
Ultimate Strength in tension, σ _{uts} (ksi)	115.000	183.000
Ultimate Strength in compression, σ_{ucs} (ksi)	115.000	363.000
Density (lb/in^3)	0.162	0.072

 Table 2 (Davidson, AEE 471 Handout, 8/31/18)
 Paint 1/18

For initial constraints, stuts AC and CD are to be I-beams with a maximum outer depth of four inches. Struts AB and DE are to have thin-walled circular cross sections with a maximum outer radius of four inches. Finally, struts BC and CE are to have square or rectangular cross sections featuring a maximum outer depth of six inches.

LIMIT AND DESIGN LOADS

First in this design process comes calculating limit and design loads. These will later lead to calculating appropriate factors of safeties of various failure modes. Limit loads are the internal forces experienced by each strut for each load case. The maximum limit load will be picked from the appropriate load case, and this will continue to be a driving force for further design conditions as the design develops. Design loads will be calculated, by multiplying the limit loads by a factor of safety of 1.5, or in the case of a yield stress will be a 1.25 factor of safety. Results for this aspect of the design are shown below in *Tables 3 and 4*. Design loads and peak limit loads are in *Table 4*, with the limit loads for each load case are in *Table 3*. Calculations for this are shown in Appendix A, where numbers are calculated in Appendix m in MATLAB.

		No Gimbal (Case 1)	Gimbal Right (Case 2)	Gimbal Left (Case 3)
Load Cases and Limit Loads	P1 (lbf)	101,196	120,000	88,391
	P2 (lbf)	227,655	227,655	227,655
	P3 (lbf)	104,196	88,391	120,000
	AC	50,589 (T)	58,262 (T)	42,915 (T)
	CD	50,589 (T)	42,915 (T)	58,262 (T)
Load in Strut	AB	77,211 (C)	88,922 (C)	65,499 (C)
(lbf)	DE	77,211 (C)	65,499 (C)	88,922 (C)
	BC	138,120 (C)	151,670 (C)	124,570 (C)
	CE	138,120 (C)	124,570 (C)	151,670 (C)

Table 3 (Appendix A and M)

Table 4 (Appendix A and M) (T=Tension, C=Compression)

Strut	Load Case	Peak Limit Load (lbf)	Design Load (lbf)
AC	Gimbal Right (2)	58,262 (T)	87,392 (T)
CD	Gimbal Left (3)	58,262 (T)	87,392 (T)
AB	Gimbal Right (2)	88,922 (C)	133,380 (C)
DE	Gimbal Left (3)	88,922 (C)	133,380 (C)
BC	Gimbal Right (2)	151,670 (C)	227,500 (C)
СЕ	Gimbal Left (3)	151,670 (C)	227,500 (C)

DESIGN CONDITION ONE

-Appendix B, C, and D, Appendix M for calculations in MATLAB (lines 70-209)

This section utilizes all Titanium Struts, utilizing the material properties available in Table 2.

Total Weight (Design Condition 1 Only) = 620.837 lbs Given By Equation 1: Total Weight = $2 * Weight_{BC,CE} + 2 * Weight_{AB,DE} + 2 * Weight_{AC,CD}$ (1)

STRUTS AB and DE, CIRCULAR CROSS SECTIONS (Tubes)

-Appendix B, Appendix M for calculations (lines 118-160) -In Compression





Table 5

Requirements		
Limit Load (lbf)	88,922.000	
Design Load (lbf)	133,380.000	
Desiş	gn Geometry	
Outer Radius (in)	4.000	
Inner Radius (in)	3.811	
Length of strut (in)	201.156	
Analysis Information		
Cross-sectional area (in ²)	4.638	
Moment of Inertia I (in ⁴)	35.391	
Correlation coefficient (y)	0.810	
Cylindrical Buckling coefficient k_x	28,729.000	
Failure Predictions		

Critical Load-Euler Buckling (lbf)	133,800.000	
Critical Load-Cylindrical Buckling (lbf)	1,682,700.000	
Critical Load-Ultimate Failure (lbf)	533,350.000	
Critical Load-Yielding (lbf)	510,170.000	
Factors of Safety		
Euler Buckling	1.505	
Cylindrical Buckling	18.924	
Ultimate Failure	5.998	
Yielding	5.737	
Weight of each Strut (lbs)	151.135	

STRUTS AC and CD, I-BEAM CROSS SECTIONS

-Appendix C, Appendix M for calculations (lines 70-117) -In Tension

Figure 3



Table 6		
Requirements		
Limit Load (lbf)	58,262.000	
Design Load (lbf)	87,392.000	
Design Geometry		
bf-bottom (in)	1.037	
bf-top (in)	1.100	
tf-bottom (in)	0.400	
tf-top (in)	0.400	
tw	0.200	
bw	0.500	
Length of strut (in)	72.000	
Analys	is Information	
Cross-sectional area (in^2)	0.795	
Moment of Inertia I (in ⁴)	0.019	
Failur	re Predictions	
Critical Load-Ultimate Failure (lbf)	91,402.000	
Critical Load-Yielding (lbf)	87,428.000	
Factors of Safety		
Ultimate Failure	1.569	
Yielding	1.501	
Weight of Each Strut (lbs)	9.270	

STRUTS BC and CE, SQUARE/RECTANGULAR CROSS SECTIONS

-Appendix D, Appendix M for calculations (lines 161-209) -In Compression

Figure 4



Requirements		
Limit Load (lbf)	151,670.000	
Design Load (lbf)	227,500.000	
Desig	gn Geometry	
Base (in)	5.700	
Height (in)	5.700	
Thickness (in) Equivalent Thickness Used	0.180	
Length of strut (in)	232.992	
Analys	is Information	
Cross-sectional area (in^2)	3.974	
Moment of Inertia I (in ⁴)	20.205	
Buckling coefficient k_h	4.000	
Failure Predictions		
Critical Load-Euler Buckling (lbf)	227,760.000	
Critical Load-Crippling (lbf)	349,750.000	
Critical Load-Local Buckling	241,840.000	
Critical Load-Ultimate Failure (lbf)	457,060.000	

Critical Load-Yielding (lbf)	437,180.000	
Factors of Safety		
Euler Buckling	1.502	
Crippling	2.306	
Local Buckling	1.595	
Ultimate Failure	3.014	
Yielding	2.883	
Weight of Each Strut (lbs)	150.013	

DESIGN CONDITION TWO

-Appendix E, Appendix M for calculations in MATLAB (lines 281-336) This section utilizes all Titanium Struts, utilizing the material properties available in Table 2.

Total Weight (Design Condition 2 Only) = 889.029 lbs

Given By Equation 2:

 $Total Weight = 2 * Weight_{BC, CE_{NEW}} + 2 * Weight_{AB, DE} + 2 * Weight_{AC, CD}$ (2)

Design Condition Two, Deflection		
Deflection BC, CE (in) (new dimensions)	0.303	
Deflection AB, DE (in) (original dimensions)	0.249	
Deflection - Y BC,CE (in) (new dimensions)	0.249	
Deflection - Y AB,DE (in) (new dimensions)	0.237	
Factor of Safety BC, CE (new dimensions)	1.503	
Factor of Safety AB, DE (original dimensions)	1.579	
New Information For BC, CE Struts		
Base (in)	6.000	
Height (in)	6.000	

Table 8

Thickness (in) Equivalent Thickness Used	0.332			
Cross-sectional area (in^2)	7.527			
Moment of Inertia I (in ⁴)	40.441			
Buckling coefficient k_h	4.000			
Failure Predictions New BC, CE				
Critical Load-Euler Buckling (lbf)	455,860.000			
Critical Load-Crippling (lbf)	662,390.000			
Critical Load-Local Buckling	1,477,800.000			
Critical Load-Ultimate Failure (lbf)	865,620.000			
Critical Load-Yielding (lbf)	827,980.000			
Factors of Safety				
Euler Buckling	3.006			
Crippling	4.367			
Local Buckling	9.744			
Ultimate Failure	5.707			
Yielding	5.459			
Weight of Each Strut (lbs)	284.108			

DESIGN CONDITION THREE

-Appendix F, Appendix M for calculations in MATLAB (lines 210-280) This section utilizes Titanium and Boron Struts, utilizing the material properties available in Table 2.





Requirements			
Limit Load (lbf)	151,670.000		
Design Load (lbf)	227,500.000		
Design Geometry			
bti (in)	5.775		
tti (in)	0.120		
tbo (in)	0.120		
bbo (in)	5.175		
Length of strut (in)	232.992		
Analysis Information			
Cross-sectional area (in ²)	7.516		
Moment of Inertia I (in ⁴)	42.154		
Buckling coefficient k_h	4.000		
Failure Predictions			
Critical Load-Euler Buckling (lbf)	475,170.000		
Critical Load-Crippling (lbf)	661,380.000		
Critical Load-Local Buckling	1,340,600.000		

Critical Load-Ultimate Failure (lbf)	864,300.000		
Critical Load-Yielding (lbf)	826,720.000		
Deflection (in)	0.303		
Factors of Safety			
Euler Buckling	3.133		
Crippling	4.361		
Local Buckling	8.839		
Ultimate Failure	5.699		
Yielding	5.451		
Deflection	1.500		
Weight			
Titanium (lbs)	104.628		
Boron (lbs)	41.670		
Total Strut Weight (lbs)	146.298		

FINAL DESIGN INFORMATION

Tuble 10					
Strut	Critical Failure Mode	Factor of Safety	Weight per Strut (Ibs)		
AC, CD	Yielding	1.501	9.270		
AB, DE	Euler Buckling	1.505	151.135		
BC, CE	Deflection	1.500	146.298		
TOTAL STRUCTURE	-	-	613.410		

Table 10