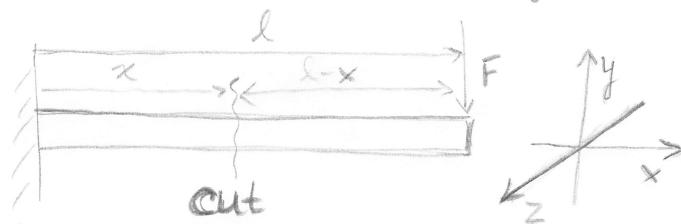


# Derivations of Lab 3 Background - MAE 315

## Elementary Beam theory



→ Derive the moment in the beam

$$\begin{array}{l}
 \text{Free Body Diagram:} \\
 \text{Force } F \text{ acts downwards at } x = l. \\
 \text{Reaction force } +F \text{ acts upwards at } x = 0. \\
 \text{Distance from reaction to cut: } l-x. \\
 \text{Bending moment: } M \text{ acts clockwise at the cut.} \\
 \text{Equilibrium equation: } -M - F(l-x) = 0 \\
 M = -F(l-x)
 \end{array}$$

## Bending

$$\frac{d^2y}{dx^2} = \frac{M(x)}{EI(x)}$$

No taper -  $I$  is constant along beam  
 \* Drop negative sign in  $M(x)$  w/ the understanding the deflection ( $y$ ) is downward.

$$\frac{d^2y}{dx^2} = \frac{+F(l-x)}{EI}$$

## Integrate

$$\frac{dy}{dx} = \frac{F}{EI} \left( lx - \frac{x^2}{2} \right) + C_1$$

○ slope of beam at the wall is zero!

(2)

Integrate Again

$$\frac{dy}{dx} = \frac{F}{EI} \left( lx - \frac{x^2}{2} \right)$$

$$y = \frac{F}{EI} \left( \frac{lx^2}{2} - \frac{x^3}{6} \right) + C_2$$

Deflection at the wall is zero!

Deflection at the end of the beam,  $y(x=l) = S$

$$S = \frac{Fl^3}{3EI}$$

$$\rightarrow S \left( \frac{3EI}{l^3} \right) = F \rightarrow K_{eq} = \frac{3EI}{l^3}$$

Therefore,

$$y(x) = S \left( \frac{3x^2}{2l^2} - \frac{x^3}{2l^3} \right)$$

"Equivalent Spring Stiffness"

"Deflection along the beam"

→ Diff. w/r to time

$$\dot{y}(x) = \dot{S} \left( \frac{3x^2}{2l^2} - \frac{x^3}{2l^3} \right)$$

$\dot{S}$  - Velocity of the end of the beam.

Find the equivalent Mass



→ Conserve Kinetic Energy

$$\frac{1}{2} M_{eq} \dot{S}^2 = \rho A \int_0^l \frac{1}{2} (\dot{S})^2 \left( \frac{3x^2}{2l^2} - \frac{x^3}{2l^3} \right)^2 dx$$

- $\rho$  - Density of the beam
- $A$  - cross sectional area

(3)

$$M_{eq} = \rho A \int_0^l \left( \frac{9}{4} \frac{x^4}{l^4} - \frac{3x^5}{2l^5} + \frac{x^6}{4l^6} \right) dx$$

$$M_{eq} = \rho A \left( \frac{9}{20} \frac{x^5}{l^4} - \frac{3x^6}{12l^5} + \frac{x^7}{28l^6} \right) \Big|_0^l$$

$$M_{eq} = \rho Al \left( \frac{9}{20} - \frac{1}{4} + \frac{1}{28} \right)$$

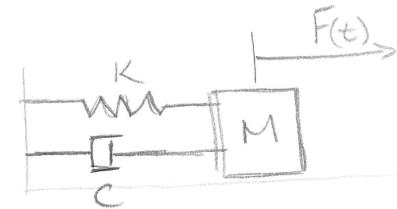
$$M_{eq} = M_{beam} \left( \frac{1}{4} \right) \left( \frac{9}{5} + \frac{1}{7} - 1 \right)$$

$$M_{eq} = M_{beam} \left( \frac{1}{4} \right) \left( \frac{63 + 5 - 35}{35} \right)$$

$$M_{eq} = \frac{33}{140} M_{beam}$$

"Equivalent Mass of the beam"

# Solving Differential Equations Lab 3 - MAE 315



Governing Egu.

$$M\ddot{x} + C\dot{x} + Kx = F(t)$$

$$\ddot{x} + 2\zeta\omega_n \dot{x} + \omega_n^2 x = \frac{1}{m} F(t)$$

$$2\zeta\omega_n = \frac{C}{m}$$

$$\omega_n^2 = \frac{K}{m}$$

→ Sinusoidal Forcing  $F(t) = F_0 \sin(\omega t)$

Suppose  $x = X \sin(\omega t - \delta) = \underbrace{X \cos \delta \sin \omega t}_A - \underbrace{X \sin \delta \cos \omega t}_B$

$$\dot{x} = Aw \cos \omega t - Bw \sin \omega t$$

$$\ddot{x} = -Aw^2 \sin \omega t - Bw^2 \cos \omega t$$

→ Substitute:

$$-Aw^2 \sin \omega t - Bw^2 \cos \omega t + 2\zeta\omega_n(Aw \cos \omega t - Bw \sin \omega t) + \omega_n^2(A \sin \omega t + B \cos \omega t) = \frac{F_0}{m} \sin \omega t$$

→ Equate Coefficients

$$-Aw^2 - 2\zeta\omega_n w B + \omega_n^2 A = \frac{F_0}{m}$$

$$-Bw^2 + 2\zeta\omega_n w A + \omega_n^2 B = 0$$

Solve for A, B

$$B(\omega_n^2 - \omega^2) = -25\omega_n\omega A$$

$$B = \frac{-25\omega_n\omega A}{\omega_n^2 - \omega^2}$$

$$A(\omega_n^2 - \omega^2) - 25\omega_n\omega B = \frac{F_0}{m}$$

$$A(\omega_n^2 - \omega^2)^2 + (25\omega_n\omega)^2 A = \frac{F_0}{m} (\omega_n^2 - \omega^2)$$

$$A = \frac{\frac{F_0}{m} (\omega_n^2 - \omega^2)}{(\omega_n^2 - \omega^2)^2 + (25\omega_n\omega)^2} \quad \left. \right\} A = X \cos \delta$$

$$B = \frac{-25\omega_n\omega \left( \frac{F_0}{m} \right)}{(\omega_n^2 - \omega^2)^2 + (25\omega_n\omega)^2} \quad \left. \right\} -B = X \sin \delta$$

$$X = \sqrt{A^2 + B^2} = \frac{F_0}{m} \sqrt{\frac{(\omega_n^2 - \omega^2)^2 + (25\omega_n\omega)^2}{(\omega_n^2 - \omega^2)^2 + (25\omega_n\omega)^2}}$$

$$X = \frac{F_0/m}{\sqrt{(\omega_n^2 - \omega^2)^2 + (25\omega_n\omega)^2}}$$

$$\tan \delta = -\frac{B}{A} = \frac{25\omega_n\omega}{(\omega_n^2 - \omega^2)}$$

# Laplace Solution

$$F(t) = F_0 S(t)$$

$$\dot{X}(0) = 0$$

$$X(0) = 0$$

$$\mathcal{L}(\ddot{X} + 2\zeta\omega_n \dot{X} + \omega_n^2 X)$$



$$s^2 X + 2\zeta\omega_n s X + \omega_n^2 X = \mathcal{L}\left(\frac{1}{m} F_0 S(t)\right) = \frac{F_0}{m}$$

$$X = \frac{F_0/m}{(s^2 + 2\zeta\omega_n s + \omega_n^2)}$$

→ Complete the squares:  $s^2 + 2\zeta\omega_n s + \omega_n^2 + \zeta^2\omega_n^2 - \zeta^2\omega_n^2$

$$X = \frac{F_0/m}{(s + \zeta\omega_n)^2 + (1 - \zeta^2)\omega_n^2} \cdot \frac{\sqrt{1 - \zeta^2} \omega_n}{\sqrt{1 - \zeta^2} \omega_n}$$

$$X = \frac{F_0/m}{\sqrt{1 - \zeta^2} \omega_n} \frac{\sqrt{1 - \zeta^2} \omega_n}{(s - -\zeta\omega_n)^2 + (1 - \zeta^2)\omega_n^2}$$

$\underbrace{\qquad\qquad\qquad}_{\frac{b}{(s-a)^2 + b^2}}$    |||    $b = \sqrt{1 - \zeta^2} \omega_n$   
 $a = -\zeta\omega_n$

$X(t) = \frac{F_0/m}{\sqrt{1 - \zeta^2} \omega_n} e^{-\zeta\omega_n t} \sin(\sqrt{1 - \zeta^2} \omega_n t)$
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